

IN THE SPECIFICATION

Please replace the section entitled DETAILED DESCRIPTION OF THE DRAWINGS with the following section shown in marked-up form:

DETAILED DESCRIPTION OF THE DRAWINGS

[0025] Figure 1 depicts the arrangement of a valve drive in a cylinder head 2 of an internal combustion engine for the purpose of the activation of a gas exchange valve 4 positioned on the intake side or the outlet side. The cylinder head 2, which is depicted in cross-section, has, for this purpose, a valve accommodation boring 3 for guiding and sealing off the gas exchange valve 4. The gas exchange valve 4 is designed as a disk valve, which is concentrically oriented, with its valve seat surface, towards the valve seat positioned in the intake- or outlet channel.

[0026] An electromagnetic actuator is provided as a valve drive above the gas exchange valve 4 in the plane of the gas exchange valve, which is also depicted in cross-section in a perspective view, within the stator 1 of which [actuator] an axially movable rotor 12 is positioned, which [rotor] is connected detachably with the valve stem 7 of the gas exchange valve 4 by way of a coupling element 17. This valve drive, which is designed as a linear motor, guarantees a variable gas exchange in which the point in time of opening of the valve, the stroke of the valve, as well as the duration of opening of the gas exchange valve 4 can be adjusted as desired in dependence on the triggering of several current coils 18 positioned in parallel in the stator 1.

[0027] The rotor 12 forms, along with the stator 1, an independently operable structural component which is preferably functionally preinspectable, and is detachably connected with the gas exchange valve 4. The coupling element 17 positioned between the rotor 12 and the gas exchange valve 4, which [element] produces a force-locking and/or form-locking connection between the rotor 12 and the gas exchange valve 4, is necessary for this purpose.

[0028] As is evident from Figure 1, the stator 1 is coaxially oriented and attached to the rotor 12 and the coupling element 17 attached to the rotor 12, opposite the gas exchange valve 4 in the cylinder head 2. For the space-saving integration of the coupling element 17 between the valve accommodation boring 3 (valve stem guide of the gas exchange valve 4) and the support surface of the stator 1, a gradated boring 19 is provided in the cylinder head 2. An auxiliary spring can, if needed, be positioned between the coupling element 17 and the base of the gradated boring 19 in order, for the

prevention of contact with the piston, to be able to securely close the gas exchange valve 4 again in the event of a failure of the current coils 18.

[0029] The rotor 12 is designed as a narrow rotor plate 11, into which [plate] several magnetic parts 21, which are concentrically positioned one above the other and which have an alternating magnetic orientation, are inserted. The magnetic parts 21 are positioned in a radial air gap in the tooth area 20 of the stator 1 positioned on both sides of the rotor plate 11, which [teeth-area] has two teeth in the internal area of the stator plates 9b, 9c between the current coils 18, which teeth align with one another, are oriented towards the level magnetic parts 21, and are positioned linearly to the rotor plate 11. The arrangement selected guarantees, independently of the number of the teeth, that the magnetic parts 21 positioned in several rows in the rotor plate 11 always align, corresponding to their magnetic orientation, with the coordinated teeth of the stator plates 9b, 9c. The construction of the stator 1 described above provides a multiplicity of current coils 18 on both sides of the rotor plate 11 between the four stator plates 9a, 9b, 9c, 9d oriented transversely to the rotor plate 11, whereby the current coils 18 can advantageously also be triggered electrically, independently of one another.

[0030] The stator 1 is constructed in such a manner that two stator plates 9b, 9c, which are identical in construction and have their tooth areas 20 positioned one above the other, are always stacked in a manner aligning with one another and are separated from one another by means of spacers 10. The first stator plate 9a, which is supported on the cylinder head 2 as a base plate and forms the ground frame, is distinguished from the three stator plates 9b, 9c, 9d placed above it by its vertical bushing 8 for the rotor 12, which [bushing] is slot-shaped in cross-section, the rotor plate 11 of which [reter] extends into the tooth area 20 of both stator plates 9b, 9c, which function as pole shoes. Above the rotor 12 and the current coil 18, the fourth stator plate 9d forms, to a certain extent, the closing frame of the stator 1.

[0031] Figure 1 consequently depicts a valve drive for a gas exchange valve in a power engine or processing engine, the magnetic rotor 12 of which, along with a rotor section placed at a distance from the gas exchange valve, extends in a longitudinally movable manner inside a bushing 8 of a stator 1 provided with several current coils 18. In accordance with the invention, the rotor 12 is designed, in the area of the rotor section, as a rotor plate 11 moved vertically in the stator 1, preferably in the manner of a flat-body slide valve, within which [plate] several magnetic parts 21 are oriented in several planes.

[0032] At least one bushing 8 adjusted to the cross-sectional contour of the rotor plate 11, through which [bushing] a rotor bar 14 connecting the rotor plate 11 with the coupling element 17 extends, is provided in the base area of the stator 1, which is oriented to the gas exchange valve 4. The base area of the stator 1 is designed particularly simply, in terms of production technology, as a

rectangular and level first stator plate 9a, which has the bushing 8 for the rotor bar 14 positioned centrally on its longitudinal axis. On both sides of the bushing 8, the necessary current coils 18, onto which the second level stator plate (9b) is applied, can be arranged particularly simply in successive rows on the first stator plate 9a, which [plate] has at least one second bushing 8 in the tooth area (20) aligning with the first bushing 8, which [second-bushing] is adjusted to the cross-section of the rotor plate 11. Above the side of the second stator plate 9b oriented away from the current coils 18, the third stator plate 9c, which is constructed identically with the second stator plate 9b, is positioned aligning with the second stator plate 9b. The third stator plate 9c is separated from the second stator plate (9b) by at least one pair of non-magnetic spacers 10. On the third stator plate 9c, several current coils 18 are, in a manner analogous to the current coils 18 positioned below the second stator plate 9b, likewise placed in a series on both sides of the rotor plate 11, to which the fourth stator plate 9d is fixed. Each current coil 18 is penetrated by a rod-shaped magnetic core 15 which, for the closing of the magnetic circuit with its ends, contacts the stator plates 9a, 9b or 9c, 9d, respectively, corresponding to the current coils 18.

[0033] Figure 2 presents a lateral view of the valve drive depicted in Figure 1, from which it is evident that the first stator plate 9a is, in the present example of implementation, provided with three bushings 8, which are adjusted to the contour of the three rotor bars 14 in a manner corresponding to the clearance. The rotor bars 14 projecting through the bushings 8 are, at approximately the level of the second stator plate 9b, brought together, over a full surface, to the area of the rotor plate 11 supporting the magnetic parts 21, the magnetic parts 21 of which [plate] are, in the lateral view in accordance with Figure 2, hidden by the current coils 18, the second and third stator plate 9b, 9c, as well as by the spacers 10 located between the same. Below the first stator plate 9a, the rotor bars 14 are likewise brought together to a solid connecting section 22, which accommodates the coupling element 17.

[0034] Figure 3 depicts, in a manner diverging from Figure 2, the rotor bars 14 integrally formed with both ends of the connecting sections 22, which [rotor-bars] encompass the first stator plate 9a in a manner corresponding to the clearance in the direction of the area of the rotor plate 11 that has the magnetic parts 21. The rotor plate 11 has an oblong slot 6 between the rotor bars 14. The oblong slot 6 accommodates the first stator plate 9a, adjusted to the length and the thickness of the stator plate 9a in a manner corresponding to clearance. The clearance existing between the thickness of the stator plate 9a and the oblong slot 6 is at least as great as the work stroke of the rotor 12. This has the advantage that the bushings 8, which are known from Figures 1, 2, are not necessary in the first stator plate 9a. The construction of the valve drive in accordance with Figure 3 otherwise corresponds to the previous details explained in Figures 1, 2.

[0035] Figure 4, diverging from the previous explanations of the object of the invention, depicts, instead of the use of a multiplicity of cylindrical current coils 18 positioned next to one another, the use of only one pair of oval current coils 18 positioned one above the other on each side of the rotor plate, so that, instead of the rod-shaped magnetic cores in the cylindrical current coils 18, the magnetic cores 15, corresponding to the oval shape of the current coils 18, now fill up the intermediate space within each oval current coil 18.

[0036] As is evident from Figure 5, the second and third stator plate 9b, 9c can, in the event of desire or need, be designed in a multi-part manner, and even displaced in their planes, so that multi-pole stator plates with relatively little space requirement can be produced.

[0037] Figure 6, starting from the basis of the descriptions to Figures 1 to 5, depicts a view from above of the first stator plate 9a forming the ground frame, which [stator-plate] is supported on the cylinder head 2 as a base plate, which is only partially depicted in a rough manner in the present view in the area of its slot-shaped bushing 8. This stator plate 9a accommodates a pair of guide elements 13a over a partial span on both sides of the narrow rotor plate 11 depicted in the view from above, either directly on the rotor plate 11 or in the area of the rotor bar 14, which [guide-elements] are inserted into grooves 5 of the stator plate 9a. A particularly simple but nevertheless precise and clamping-free guidance of the rotor 12 in the stator 1 is achieved by this means.

[0038] Figure 7, in contrast to Figure 6, depicts a multiple supporting and guidance of the rotor 12 inside the bushing 8 of the first stator plate 9a, for which the guide elements 13a, 13b are, on both of the external sections of the rotor 12, positioned on both sides of the plate-shaped rotor 12 in grooves 5 of the stator plate 9a.

[0039] Figure 8, in supplement to Figures 1 to 7, depicts in a lateral view the partially depicted rotor plate 11, with several magnetic parts 21 which are positioned in several series of magnets X1, X2 of the rotor plate 11 positioned in parallel, one above the other, whereby the series of magnets X1, X2 are inclined opposite to the horizontal orientation of the tooth area 20.

[0040] In an alternate manner to Figure 8, the magnetic parts 21 are accommodated, in Figure 9, horizontally in the rotor plate 11 in several series of magnets X1, X2 positioned in parallel one above the other, while the stator plates 9b, 9c provided with the tooth areas 20 are inclined opposite the series of magnets X1, X2.

[0041] The constructions proposed in Figures 7 and 8 improve the transition of the electromagnetic power flux during the relative motion of the rotor 12 in the stator 1, as the result of which a more efficient utilization of the magnetic force results.

[0042] In summary, the valve drive proposed in accordance with the invention is distinguished by the following characteristics:

1. Through the configuration of the rotor 12 in accordance with the type of a flat-body slide valve, an extremely narrow constructional width results for the valve drive, so that this can be integrated into every cylinder head without any problem.
2. The magnetic circuit can be optimally adjusted to the performance requirements of the valve drive through the fact that the number, the construction, and the electrical triggering of the current coils 18 can be varied in a simple construction and in accordance with preference. Through the simple geometry of the stator, which is brought about through the use of the stator plates 9a-d presented, among other ways, not only cylindrical, but also oval current coils 18 can be used. Upon the use of several cylindrical current coils 18, a particularly small diameter of coil can be achieved, so that the expenditure for copper, as well as the influence on the level of efficiency in the current circuit that is connected with the same, is minimal. The actuator can consequently be operated with a lower electrical voltage. The current coils can, depending on the characteristic operating curve that is desired, each be flowed through by electrical current in parallel, in series, or in sequence, through which the possibility exists for the recovery of electrical energy from the electrical motor power specifically prevailing within the magnetic circuit. Furthermore, only a slight dispersion of the magnetic field results.
3. A good cooling of the coil, and a particularly simple mounting as well as dismounting of the current coils 18, results from the fact that the current coils 18 are positioned exclusively on the side of the rotor 12.
4. A guidance of the rotor 12 independently of thermally-induced changes in geometry results, whereby thermal expansions between the rotor 12 and the stator 1 have no influence on the guidance. Through the use of guide elements 13a, 13b that is hereby proposed, the rotor 12 is guided securely and supported against the high magnetic transverse forces acting there, even in a critical area of air gap, as well as against transverse acceleration forces. The number of guide elements that is thereby used can vary between two and a multiple thereof.

[0043] The invention proposed consequently guarantees:

- Economic manufacturing tolerances;
- Economic mounting and automatic adjustment of the valve drive;
- Low losses in the magnetic circuit;
- High efficiency, since the valve drive is optimally adjustable and has only low frictional forces;

- Thermally stable operation of the valve drive, even during the run-up phase and the cooling phase of the engine;
- Simple workshop service.

List of references:

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| 1 | Stator |
| 2 | Cylinder head |
| 3 | Valve accommodation boring |
| 4 | Gas exchange valve |
| 5 | Groove |
| 6 | Oblong slot |
| 7 | Valve stem |
| 8 | Bushing |
| 9a-d | Stator plate |
| 10 | Spacer |
| 11 | Rotor plate |
| 12 | Rotor |
| 13a | Guide element |
| 13b | Guide element |
| 14 | Rotor bar |
| 15 | Magnetic core |
| 16 | Clamping ring |
| 17 | Coupling element |
| 18 | Current coil |
| 19 | Gradated boring |
| 20 | Tooth area |
| 21 | Magnetic part |
| 22 | Connecting section |